

# Experimental Investigation Of Performance And Emission Characteristics Of Tamanu Oil As Alternative Fuel In Ci Engine

G. Deepankumar

E- Mail: deepanleo@gmail.com

PG Scholar, Saranathan College of Engineering, Trichy, Tamilnadu.

**Abstract** - The increasing industrialization and motorization of the world has led to a steep rise for the demand of petroleum products. Petroleum based fuels are obtained from limited reserves. These finite reserves are highly concentrated in certain regions of the world. Therefore, those countries not having these resources are facing a foreign exchange crisis, mainly due to the import of crude oil. Hence, it is necessary to look for alternative fuels, which can be produced from materials available within the country. Biodiesel can be extracted from vegetable oils and waste fats. The present work examined the use of a Tamanu oil, a new possible source of alternative fuel for direct injection diesel engine. The biodiesel has been prepared from Tamanu oil by Trans-Esterification method. Diesel and Tamanu oil methyl ester (B15, B25, B50, B100) fuel blends are used for conducting the performance and emission in terms of 20% load increments from no load to full load. From the experiments it is found that Tamanu oil is a better NOx reduction fuel.

**Keywords:** Diesel, Tamanu oil, Trans-esterification, performance and Emission.

## 1. Introduction

The idea of using vegetable oil as a substitute for diesel fuel was demonstrated by the inventor of the diesel engine, Rudolph Diesel, around the year 1900, when vegetable oil was proposed as fuel for engines. The oil use as diesel fuel was limited due to its high viscosity (near 10 times of the gas oil). In order to adapt the fuel to the existing engines the properties of vegetable oil had to be modified. Various products derived from vegetable oils have been proposed as an alternative fuel for diesel engines. ASTM International defines Biodiesel as the “mono alkyl esters of long chain fatty acids derived from renewable liquid feedstock’s, such as vegetable oils and animal fats, for use in compression ignition engines.” Biodiesel can be blended at any level with petroleum diesel to create a biodiesel blend or can be used in its pure form.

## 2. Tamanu oil

Scientific name: Calophyllum inophyllum

Other names: Alexandrian laurel, Punnai, Tamanu, Kamani.



Fig 1: Seed's shell and kernel



Fig 2: Tamanu oil

### 2.1 Advantages of Tamanu oil

1. Calophyllum inophyllum has high survival potency in nature, still productive upto 50years.
2. It does not compete with food crops.
3. Its trees serve as windbreaker at the seashore where it can reduce abrasion, protect crops and provide ecotourism and conservation of coastal demarcation.
4. It has higher oil yield than Jatropha curcas.
5. It has high heating value.

### 3. Experimental setup

Tamanu oil contains 19.58% free fatty acids. The methyl ester is produced by chemically reacting Tamanu oil with an alcohol (methyl), in the presence of catalyst (Potassium Hydrox-ide). A two stage process is used for the transesterification of Tamanu oil.

### 3.1 Reagents and materials used for experiment

1. Tamanu oil for biodiesel preparation
2. Methanol
3. Acid catalyst, sulphuric acid[H<sub>2</sub>SO<sub>4</sub>]
4. Base catalyst, Potassium Hydroxide[KOH]



Fig 3: Trans-esterification Setup

### 4. Blending of Biodiesel

The blending of oil with diesel is done by volume basis.

Table 1 : Blending of Tamanu oil with Diesel

Type of blend	Amount of diesel(ml)	Amount of biodiesel(ml)
Diesel	1000	0
B15	850	150
B25	750	250
B50	500	500
B100	0	1000

Table 2 : Properties of Diesel and Biodiesel blends

Properties	Flash Point °C	Fire Point °C	Density @ 15°C kg/m <sup>3</sup>	Calorific value KJ/Kg	Kinematic Viscosity @ 40°C cSt
Diesel	65	84	820	42000	3.12
B15	75	100	835	41300	3.24
B25	84	112	840	40600	3.32
B50	105	130	844	39900	3.75
B100	145	156	852	39000	4.25

### 5. Experimental Setup for Testing Biodiesel on Engine

The setup consists of single cylinder, four strokes, Diesel engine connected to eddy current type dynamometer for loading. Using esterified Tamanu oil in the compression ignition diesel engine at a rated speed of 1,500 rpm, the performance and emission analysis is carried out.

Table 3: Engine specification

Make	Kirlosker TV-1
Type	Vertical cylinder DI diesel engine
Number of cylinder	1
Bore x Stroke	87.5 mm x 110 mm
Compression ratio	17.5:1
Speed	1500 rpm
Rated brake power	5.2 kw
Cooling system	Water
Fuel	Diesel
Fuel injection pump	MICO inline with mechanical governer and flange mounted
Injection type	Mechanical pump-nozzle injection system
Injection pressure	220 bar
Ignition timing	23°before TDC(rated)
Ignition system	Compression ignition



Fig 4: Engine setup

### 6. Performance characteristics

#### 6.1.Brake Thermal Efficiency

Brake Thermal Efficiency is defined as break power of a heat engine as a function of the thermal input from the fuel. BTE constantly increases based on the load condition. This was due to a reduction in heat loss and increase in power with increase in percent load. The maximum brake thermal efficiency obtained is about 30% for B15, which is quite higher than that of diesel.

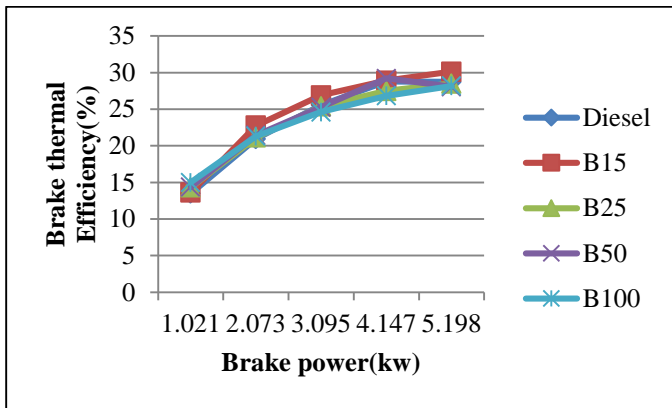


Fig 5: Brake power Vs Brake Thermal Efficiency

### 6.2 Specific Fuel Consumption

Specific fuel consumption, abbreviated SFC, compares the ratio of the fuel used by an engine to a certain force such as the amount of power the engine produces. Among the blend B15 is lowest at all loads which is normally the optimum for any diesel engine. Higher proportions of Tamanu oil in the blends increases the viscosity which in turn increases the specific fuel consumption due to poor atomization of fuel.

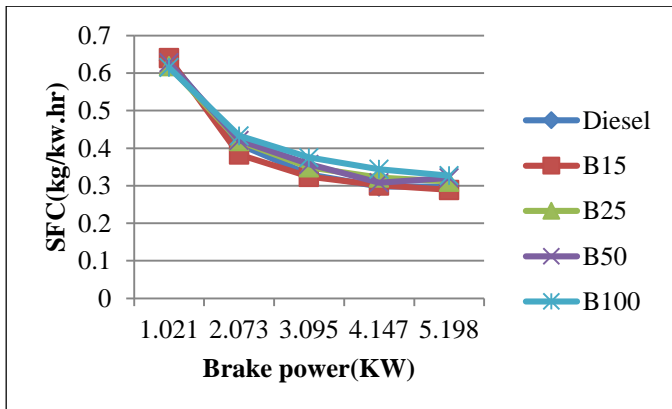


Fig 6: Brake power Vs Specific fuel consumption

### 6.3 Mechanical Efficiency

Figure 4 shows that B100 have higher efficiency at load condition. Diesel fuel has lag in the mechanical efficiency compared with that Biodiesel blends. This shows that the biodiesel lubricity have reduces the friction losses.

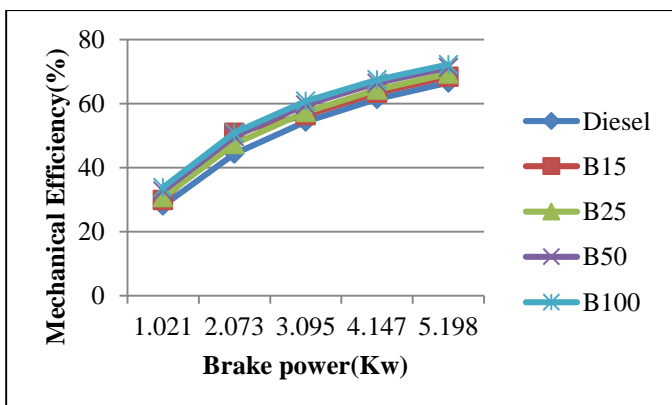


Fig 7: Brake power Vs Mechanical Efficiency

## 7. Emission characteristics

### 7.1 Smoke Emission

It is observed from the figure 6 that Smoke Density increases with increase in Load and B100 produces more smoke at all Load condition and this may be due to lesser calorific value and high density.

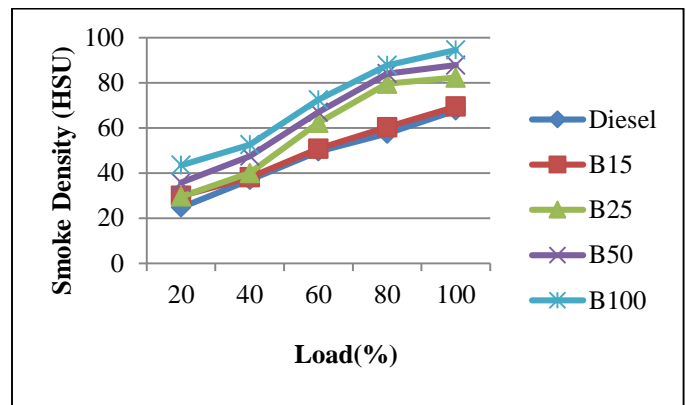


Fig 8: Load Vs Smoke Density

### 7.2 Carbon Monoxide Emission

The CO emission tend to increase with the increase in the load. It is observed that at low loads the CO emission for B15 and diesel were found to be same but at higher loads B15 produces more CO and this may be due to high viscosity and improper spray pattern with higher blend percentage resulting in incomplete combustion.

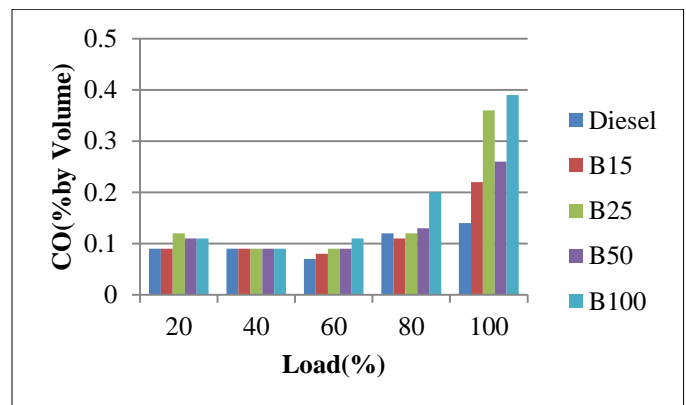


Fig 9: Load Vs Carbon Monoxide

### 7.3 Carbon Dioxide Emission

Figure 7, shows the emission levels of CO<sub>2</sub> for various blends and diesel. The Blend B15 gives less CO<sub>2</sub> emission at full load condition. The rising trend of CO<sub>2</sub> emission with load is due to the higher fuel entry as the load increases.

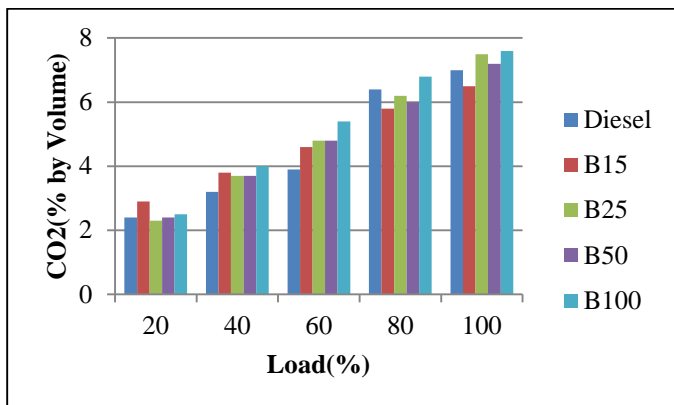


Fig 10: Load Vs Carbon Dioxide

#### 7.4 Oxygen Emission

It has been noted that Oxygen emission tend to decrease with the increase in the load and this may be due to more consumption of oxygen in the combustion chamber.

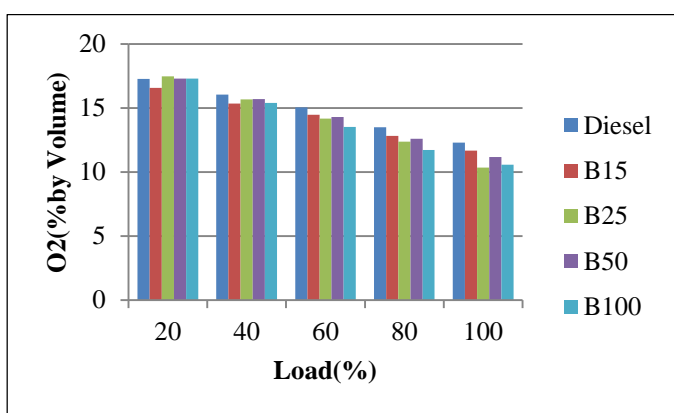


Fig 11: Load Vs Oxygen

#### 7.5 Hydro Carbon Emission

HC emission tend to increase with the increase in the load. It has been noted that almost all blends give higher HC emission than that of diesel and the reason for this may be due to poor volatility and high viscosity.

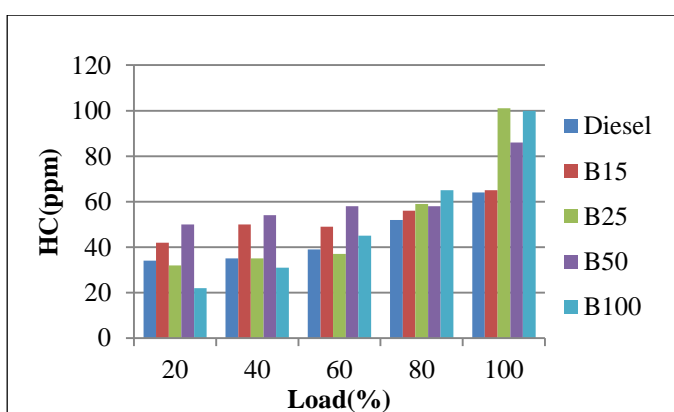


Fig 12: Load Vs Hydro Carbon

#### 7.6 Oxides of Nitrogen

The NO<sub>x</sub> emission increased as the engine load increased, due to combustion temperature increased. This proves that the emission of NO<sub>x</sub> is significantly influenced by the cylinder gas temperature and the availability of oxygen during combustion. It is observed that all blends produce lower NO<sub>x</sub> compared to diesel over the entire load range. B100 gives less

NO<sub>x</sub> over the entire load range. The reduction in NO<sub>x</sub> emission is due to reduced premixed burning rate (low heat release rate) leads to lower cylinder temperature.

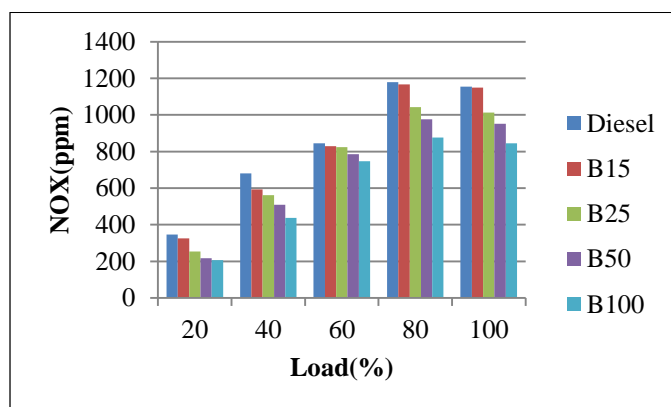


Fig 13: Load Vs Oxides of Nitrogen

#### 7.7 Exhaust Gas Temperature

The exhaust gas temperature tends to increase with increase in the brake power. The exhaust gas temperature reflects on the status of combustion inside the combustion chamber. It is observed from the graph that all blends give low exhaust gas temperature when compared to that of diesel. The diesel gives higher exhaust gas temperature because of its higher heat content.

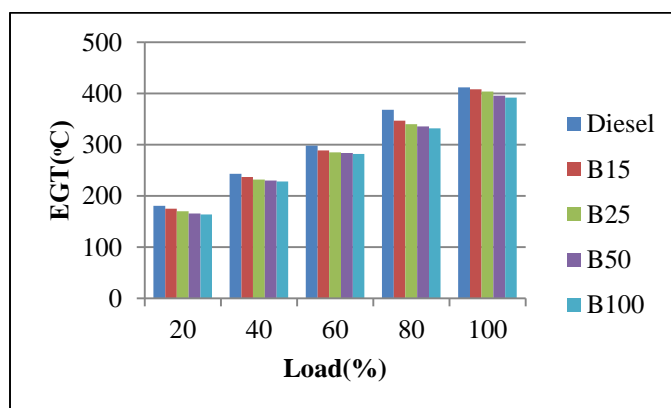


Fig 14: Load Vs Exhaust Gas Temperature

#### 8. Conclusion

The following conclusions are drawn from the investigation:

- 1) Use of a non-edible oil namely Tamanu oil is considered as a new possible source of alternative fuel for diesel engine.
- 2) No difficulty was faced at the time of starting the engine and the engine ran smoothly over the range of engine speed.
- 3) The blend B15 shows marginal increase in brake thermal efficiency.
- 4) B100 gives greater mechanical efficiency than that of diesel because of the lubricity property of Tamanu oil and reduction in friction losses.
- 5) Based on the experiment it is found that Tamanu oil is a better NO<sub>x</sub> reduction fuel as all the blends of Tamanu oil give less NO<sub>x</sub> emission when compared to that of diesel.

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